

## **Hydropower Plants: Generating and Pumping Units Series 2**

## GENERAL STUDY OF HYDRO POWERPLANT

The studied hydro-electric installation, whose layout is shown in Figure 1, is located in South America.

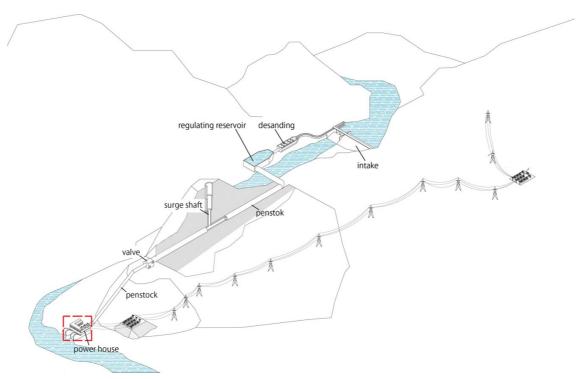


Figure 1: Layout of the hydro-electric installation

The rated discharge of the power plant Q is 50 m3 s-1. The gross head is considered as constant throughout the year and equal to 300 m. The value of the water density is  $\rho = 1'000$  kg.m-3 and the gravity acceleration is g = 9.805 m.s<sup>-2</sup>. The global efficiency of the power station (i.e. without considering the losses between B and I) is  $\eta = 0.91$ . For the purpose of simplification, the piping system is considered as a simplified penstock line, as illustrated in Figure 2. In this section, the only considered specific energy losses in the hydraulic circuit from B to I are the regular specific energy losses between III and II. Use the length of the penstock L = 5'400 m.

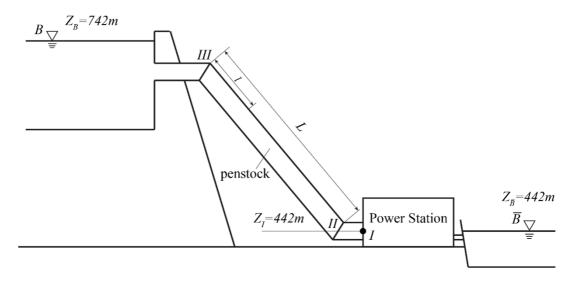


Figure 2: Simplified schematics of the hydro-electric installation

- 1) Express the potential power by using the relevant variables.
- 2) Express the distributed specific energy losses in the penstock  $gH_{r\_penstock}$  as a function of the local loss coefficient  $\lambda$ , the length of the penstock L, the penstock diameter D, and the discharge Q.
- 3) Considering the distributed specific energy losses in the penstock, express the hydraulic power  $P_h$  at point I.
- 4) The global efficiency  $\eta$  is the result of a combination of several phenomena which limit the use of the hydraulic power  $P_h$  by the hydraulic machine, thus reducing the final output power P. Provide an answer to the following questions:
  - a. Is the transferred power  $P_i$ , also called extracted power, larger or smaller than the hydraulic power  $P_h$ ? Justify your answer by giving a list of the phenomena that are responsible for this difference, and indicate which efficiency sub-terms  $\eta_i$  of the global efficiency  $\eta$  are involved.
  - b. Is the mechanical power  $P_m$  larger or smaller than the output power P? Why?
  - c. To evaluate the electrical power which is actually provided to the grid, calculating the output power P is not enough. A further efficiency subterm (which is however not included in the global efficiency  $\eta$ ) must be considered. Why?
- 5) Using the global efficiency  $\eta$ , express the output power P using the parameters introduced in questions 1) and 2).
- 6) Let's move now to the energy state of the flow discharge. Start by expressing the gauge pressure of a fluid in function of the absolute and the atmospheric pressures.
- 7) Performing an energy balance between the pressure, kinetic energy and potential energy components of the discharge flowing in the penstock, express the total gauge

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pressure  $p_{tot}(l) = p_{gauge}(l) + \frac{1}{2}\rho u^2(l)$  as a function of L, l,  $Z_I$  and  $Z_{III}$ . Consider l as the coordinate which describes the longitudinal position in the penstock (l=0 in III, and l=L in II),  $p_{gauge}(l)$  the gauge pressure and  $\frac{1}{2}\rho u^2(l)$  the dynamic pressure. Assume that the elevation in III is approximately the same as at the surface of the upper reservoir,  $Z_{III} \cong Z_B$ , and neglect the specific energy losses term.

- 8) In your opinion, could the dynamic pressure term be replaced by a constant? If yes, which geometrical characteristic must the penstock have?
- 9) Let's now put some numerical values on these results. Considering a pipe diameter D = 3 m and using the values listed in Table 2, calculate the following values:
  - a. The output power P in kW.
  - b. The electricity generation in kWh for one year.
  - c. The annual sales of the electricity (neglecting the energy losses in the generator).

Table 2. Specific values of the penstock	
parameter	value
λ	0.0100
Annual productivity	50%
Electricity Unit Price	0.06 CHF/kWh

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